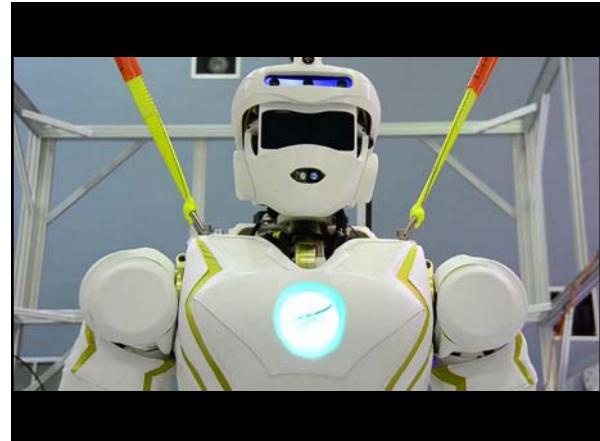


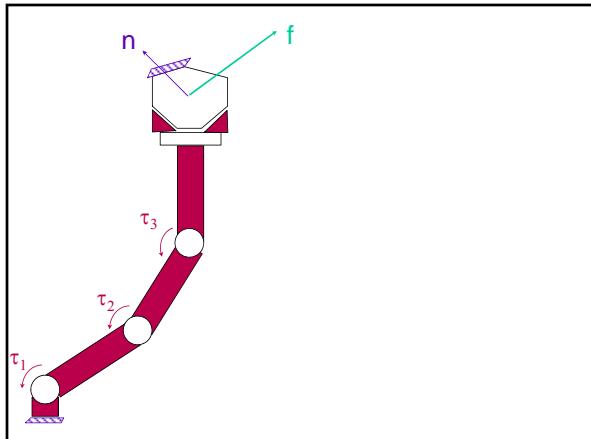
Video Segment

Valkyrie: NASA's Superhero Robot, IEEE Spectrum, 2013



Control

- Natural Systems
- PID Control
- Joint-Space Dynamic Control
- Task-Oriented Control
- Force Control



Velocity/Force Duality

$$\dot{x} = J \dot{\theta}$$

$$\tau = J^T F$$

Example (Static Forces)

$$J = \begin{pmatrix} -(l_1 S_1 + l_2 S_{12}) & -l_2 S_{12} \\ l_1 C_1 + l_2 C_{12} & l_2 C_{12} \end{pmatrix}$$

$$J^T = \begin{pmatrix} -(l_1 S_1 + l_2 S_{12}) & l_1 C_1 + l_2 C_{12} \\ -l_2 S_{12} & l_2 C_{12} \end{pmatrix}$$

$$\tau = J^T F$$

$$l_1 = l_2 = 1; \theta_1 = 0; \theta_2 = 60^\circ$$

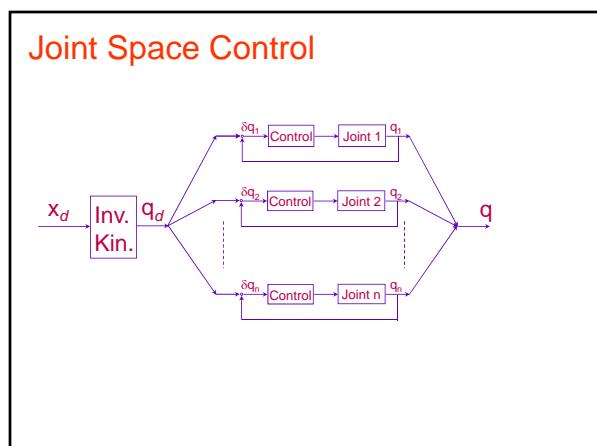
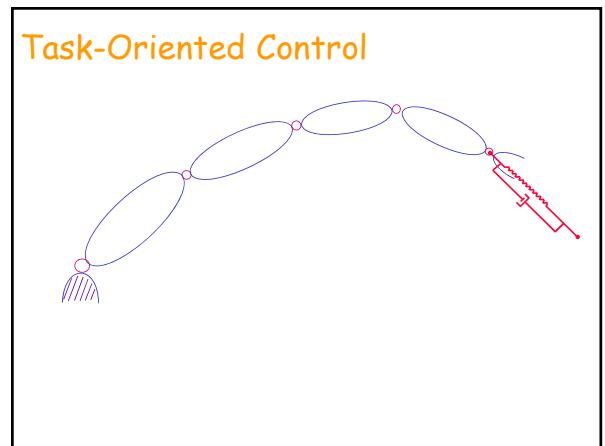
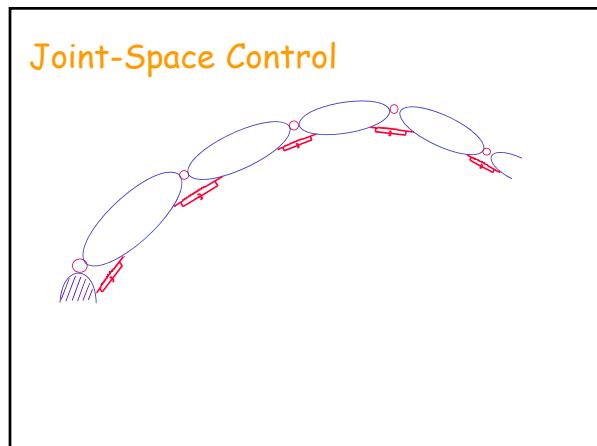
$$\tau = \begin{pmatrix} -(l_1 S_1 + l_2 S_{12}) & l_1 C_1 + l_2 C_{12} \\ -l_2 S_{12} & l_2 C_{12} \end{pmatrix} \begin{pmatrix} 0 \\ -1 \end{pmatrix} = -\begin{bmatrix} l_1 C_1 + l_2 C_{12} \\ l_2 C_{12} \end{bmatrix} = -\begin{bmatrix} 3/2 \\ 1/2 \end{bmatrix}$$

Example (Static Forces)

$$\tau = J^T F$$

$$\tau = \begin{pmatrix} -(l_1 S_1 + l_2 S_{12}) & l_1 C_1 + l_2 C_{12} \\ -l_2 S_{12} & l_2 C_{12} \end{pmatrix} \begin{pmatrix} 0 \\ -1000 \end{pmatrix} = \begin{bmatrix} l_1 C_1 + l_2 C_{12} \\ l_2 C_{12} \end{bmatrix} (-1000) = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$l_1 = l_2 = 1; \theta_1 = 90; \theta_2 = 0^\circ$$



Resolved Motion Rate Control (Whitney 72)

$$x = J(\cdot)$$

Outside singularities

$$J^{-1}(\cdot) x$$

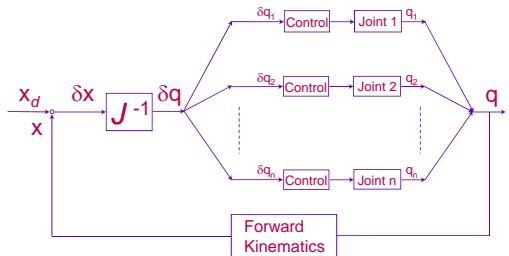
Arm at Configuration

$$x = f(\cdot)$$

$$x = x_d - x$$

$$J^{-1} x$$

Resolved Motion Rate Control



Natural Systems

Conservative Systems



$$\frac{d}{dt} \left(\frac{\partial K - V}{\partial \dot{x}} \right) - \frac{\partial (K - V)}{\partial x} = 0 \quad K = \frac{1}{2} m \dot{x}^2$$

Natural Systems

Conservative Forces



$$\frac{d}{dt} \left(\frac{\partial K}{\partial \dot{x}} \right) - \frac{\partial K}{\partial x} = - \frac{\partial V}{\partial x}$$

$$m \ddot{x} = F = -kx \quad -\frac{\partial}{\partial x} \left(\frac{1}{2} kx^2 \right)$$

Potential Energy of a spring

$$V = Work = \int_x^0 (-kx) \delta x = \frac{1}{2} kx^2$$

Natural Systems

Conservative Forces



$$\frac{d}{dt} \left(\frac{\partial K}{\partial \dot{x}} \right) - \frac{\partial K}{\partial x} = - \frac{\partial V}{\partial x}$$

Potential Energy of a spring

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Natural Systems

Conservative Systems



$$\frac{d}{dt} \left(\frac{\partial (K - V)}{\partial \dot{x}} \right) - \frac{\partial (K - V)}{\partial x} = 0 \quad K = \frac{1}{2} m \dot{x}^2$$

$$m \ddot{x} + kx = 0 \quad V = \frac{1}{2} kx^2$$

Natural Systems

Conservative Systems

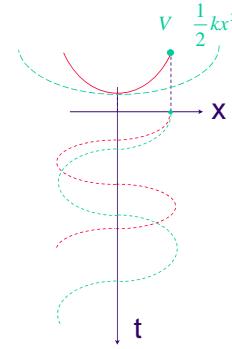
$$m \ddot{x} + kx = 0$$

Frequency increases with stiffness and inverse mass

$$\text{Natural Frequency } n = \sqrt{\frac{k}{m}}$$

$$\ddot{x} = -n^2 x = 0$$

$$x(t) = c \cos(n t)$$



Natural Systems

Dissipative Systems

$$\frac{d}{dt} \left(\frac{\partial(K-V)}{\partial \dot{x}} \right) - \frac{\partial(K-V)}{\partial x} = f_{\text{friction}}$$

Viscous friction: $f_{\text{friction}} = b\dot{x}$

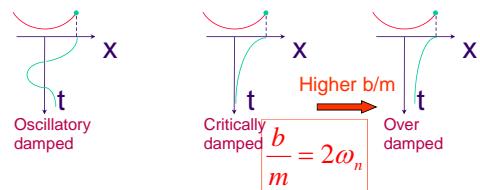
$$m\ddot{x} + b\dot{x} + kx = 0$$



Dissipative Systems

$$\frac{d}{dt} \left(\frac{\partial(K-V)}{\partial \dot{x}} \right) - \frac{\partial(K-V)}{\partial x} = m\ddot{x} + b\dot{x} + kx = 0$$

$$\ddot{x} + \frac{b}{m}\dot{x} + \frac{k}{m}x = 0$$



2^d order systems

$$m\ddot{x} + b\dot{x} + kx = 0$$

$$\ddot{x} + \frac{b}{m}\dot{x} + \frac{k}{m}x = 0$$

Diagram showing the natural frequency $\omega_n = \sqrt{\frac{k}{m}}$ and the natural damping ratio $\xi_n = \frac{b}{2\omega_n}$. A circle highlights the term $\frac{b^2}{4m^2}$.

Critically damped when $b/m = 2\omega_n$

$$\text{Critically damped system: } \xi_n = 1 \quad (b = 2\sqrt{km})$$

Time Response

$$\ddot{x} + 2\xi_n \dot{x} + \omega_n^2 x = 0$$

Natural frequency $\omega_n = \sqrt{\frac{k}{m}}$; Natural damping ratio $\xi_n = \frac{b}{2\sqrt{km}}$

$$x(t) = ce^{-\xi_n t} \cos(\omega_n \sqrt{1 - \xi_n^2} t)$$

Graph of $x(t)$ showing damped oscillations. The natural frequency is $\omega = \omega_n \sqrt{1 - \xi_n^2}$.

Example

$$\begin{array}{l} \text{Diagram: Mass } m \text{ attached to spring with stiffness } k, \text{ friction force } b. \\ \text{Equation: } m\ddot{x} + b\dot{x} + kx = 0 \\ \text{Given values: } m = 2.0, b = 4.8, k = 8.0 \\ \text{What is the "damped Natural frequency":} \\ \omega_n = \sqrt{\frac{k}{m}} = 2; \quad \xi_n = \frac{b}{2\sqrt{km}} = 0.6 \\ \boxed{2\sqrt{1 - 0.36} = 1.6} \end{array}$$

1-dof Robot Control

$$\begin{array}{l} \text{Diagram: Mass } m \text{ with force } f, \text{ position } x_0, \text{ desired position } x_d, \text{ potential field } V(x). \\ \text{Equation: } m\ddot{x} + f = 0 \\ \text{Potential Field:} \\ V(x) = 0, x < x_d \\ V(x) = 0, x > x_d \\ V(x) = \frac{1}{2}k_p(x - x_d)^2; f = -V'(x) = -\frac{V}{x} \\ \text{Equation: } m\ddot{x} = -\frac{1}{2}k_p(x - x_d)^2; m\ddot{x} + k_p(x - x_d) = 0 \\ \text{Position gain:} \end{array}$$

Passive Systems (Stability)

$$V_{goal} = \frac{1}{2} k_p (x - x_g)^T (x - x_g)$$

System $\frac{d}{dt} \left(\frac{\partial K}{\partial \dot{x}} \right) - \frac{\partial K}{\partial x} = f$
 $\Downarrow f = -\frac{\partial V_{goal}}{\partial X}$

$$\frac{d}{dt} \left(\frac{\partial K}{\partial \dot{x}} \right) - \frac{\partial (K - V_{goal})}{\partial x} = 0 \quad \text{Conservative Forces}$$

Stable

Asymptotic Stability

a system $\frac{d}{dt} \left(\frac{\partial K}{\partial \dot{x}} \right) - \frac{\partial (K - V_{goal})}{\partial x} = F_s$

is asymptotically stable if

$$F_s^T \dot{x} < 0 ; \text{ for } \dot{x} \neq 0$$

$$F_s = -k_v \dot{x} \rightarrow k_v > 0$$

Control

$$F = -k_p (x - x_{goal}) - k_v \dot{x}$$



Proportional-Derivative Control (PD)

$$m\ddot{x} \quad f \quad k_p(x - x_d) \quad k_v \dot{x}$$

$$m\ddot{x} + k_v \dot{x} + k_p(x - x_d) = 0$$

Velocity gain

Position gain

$$1. \ddot{x} + \frac{k_v}{m} \dot{x} + \frac{k_p}{m} (x - x_d) = 0$$

$$1. \ddot{x} + 2\xi\omega\dot{x} + \omega^2(x - x_d) = 0$$

$\frac{k_v}{2\sqrt{k_p m}}$ closed loop damping ratio

$\sqrt{\frac{k_p}{m}}$ closed loop frequency

Gains

$$k_p \quad m^{-2}$$

$$k_v \quad m(2^{-})$$

Gain Selection

$$\text{set } \begin{pmatrix} k_p \\ k_v \end{pmatrix} \quad \begin{matrix} m^{-2} \\ m(2^{-}) \end{matrix}$$

Unit mass system

$$k_p \quad m^{-2}$$

$$k_v \quad 2$$

m - mass system

$$k_p \quad m \cdot k_p$$

$$k_v \quad m \cdot k_v$$

Control Partitioning

$$m\ddot{x} \quad f \quad \Rightarrow \quad m(1.\dot{x}) = m f'$$

$$f = -k_v \dot{x} - k_p(x - x_d)$$

$$f = m[-k_v \dot{x} - k'_p(x - x_d)] = m f'$$

$$m\ddot{x} = m f' \quad f$$

$$1. \ddot{x} \quad f \quad \text{unit mass system}$$

$$1. \ddot{x} + k'_v \dot{x} + k'_p(x - x_d) = 0$$

$\frac{1}{2}$

$\frac{1}{2}$